

Amendments to the Specification

On page 5, please amend the fourth paragraph as follows:

This is achieved in accordance with the invention by ~~the characterizing features of claim 1~~ a hermetically encapsulated refrigerant compressor comprising a hermetically sealed compressor housing, a piston-cylinder unit disposed in an interior of the housing for compressing a refrigerant and comprising a suction valve with an intake port arranged in a valve plate of the suction valve, and a suction muffler disposed on the cylinder head of the piston-cylinder unit. The suction muffler comprises a filling volume through which the refrigerant flows to the suction valve of the piston-cylinder unit, an inlet cross section through which refrigerant flows into the suction muffler, and a compensating volume in connection with the suction muffler and the interior of the compressor housing and in which the refrigerant oscillates. The inlet cross section is simultaneously a connecting port between the compensating volume and the filling volume, and the compensating volume is formed by an outer tube which tightly encloses the intake port or the inlet cross section and encloses the refrigerant suction pipe at least along a section and is directed into the compressor housing, which suction pipe extends into the interior of the compressor

housing. The compensating volume and filling volume are arranged so that refrigerant from the suction pipe flows into the compensating volume by passing through the filling volume.

On page 5, please amend the last paragraph as follows:

It is ensured ~~by the characterizing features of claim 2~~ that sufficient compensating volume is available due to the suction pipe being guided shortly to a point shortly before the intake port in the outer tube.

On page 6, please amend paragraphs 1-5, which extend to the top of page 7, as follows:

~~The characterizing features of claim 3, namely the integral configuration of suction muffler and compensating volume, allow an especially cost-effective and rapid possibility for production.~~

By creating a compensating volume with a volume corresponding to 0.5 to 1.2 times the working volume of the piston-cylinder unit ~~according to the characterizing features of claim 4~~, it is guaranteed that the refrigerant coming from the

suction pipe will not reach the compressor housing even when the intake valve is closed and will mix there with the already heated refrigerant. It is guaranteed at the same time that during the intake process no refrigerant is drawn from the compressor housing via the compensating volume into the suction muffler or into the cylinder.

As a result of ~~the characterizing features of claim 5, which~~ ~~is~~ the creation of a compensating volume which is at least half, preferably 0.5 to 3 times the working volume of the piston of the piston-cylinder unit, the noise development following the creation of the compensating volume as a result of the flow processes into the compensating volume and into the compressor housing can be minimized in addition, so that there is no noise development which might be disturbing to the operator, which is especially important for household refrigerators. Moreover, a slightly larger compensating volume is more easy to produce from a production standpoint.

~~According to the characterizing features of claim 6 it is~~ ~~provided that the~~ The smallest flow cross section in the compensating volume has a cross-sectional surface area which corresponds to $1/4$ to $3/4$ of the cross-sectional surface area of

the intake opening. This ensures that the pressure difference becomes small, leading to a reduction in the flow losses and high noise damping to the outside.

~~According to the characterizing feature of claim 7, the~~ The cross section of the compensating volume can correspond at most to 1.5 times the piston head surface area. This ensures that on the one hand the need for space for the compensating volume will not become too large and on the other hand it is ensured that cold and warm suction gas will not mix or the boundary layer as described below will not form.

On page 8, please amend the second paragraph as follows:

~~The characterizing features of claim 8, according to which~~
In a preferred embodiment, the compensating volume has a circular cross section and the ratio of the length of the compensating volume to its diameter is higher than 10, ~~describe a preferred embodiment~~ which leads to especially low flow losses.